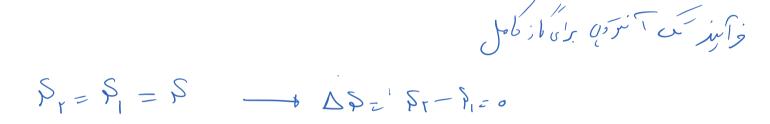
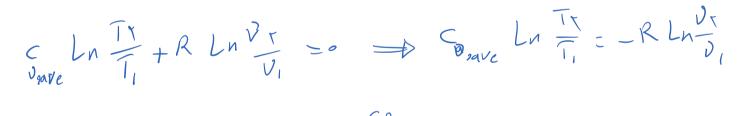
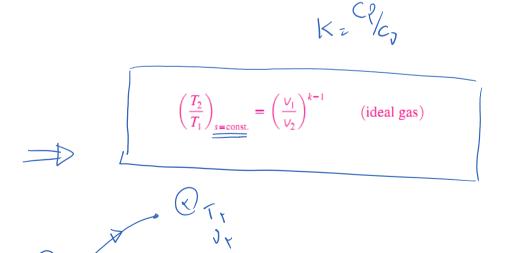
$$\underline{s_2} - \underline{s_1} = c_{v,avg} \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1} \qquad (kJ/kg \cdot K)$$
(7-33)
$$s_2 - s_1 = c_{p,avg} \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} \qquad (kJ/kg \cdot K)$$
(7-34)

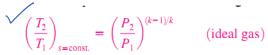
$$s_{2} - s_{1} = c_{avg} \ln \frac{T_{2}}{T_{1}}; \qquad \Rightarrow \qquad (v) = i \quad (v) =$$







برای ما زما س



(7-43)

The *third isentropic relation* is obtained by substituting Eq. 7-43 into Eq. 7–42 and simplifying:

 $\left(\frac{P_2}{P_1}\right)_{k} = \left(\frac{v_1}{v_2}\right)^k$ (ideal gas)

(7 - 44)

 $\begin{pmatrix} T_2 \\ \overline{T_1} \end{pmatrix}_{s = \text{ const.}} = \begin{pmatrix} P_2 \\ \overline{P_1} \end{pmatrix}^{(k-1)/k} = \begin{pmatrix} v_1 \\ \overline{v_2} \end{pmatrix}^{k-1}$

*ideal gas / *isentropic process VALID FOR *constant specific heats

EXAMPLE 7-10 Isentropic Compression of Air in a Car Engine

Air is compressed in a car engine from 22°C and 95 kPa in a reversible and adiabatic manner. If the compression ratio V_1/V_2 of this engine is 8, determine the final temperature of the air.

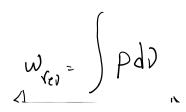
$$T_{1} = 22^{2}c + TV3 = YA8K$$

$$P_{1} = 98Kpa$$

$$\frac{T_{1}}{T_{1}} = \left(\frac{U_{1}}{U_{T}}\right)^{K-1}$$

$$\frac{T_{1}}{T_{1}} = \left(\frac{U_{1}}{U_{T}}\right)^{K-1}$$

$$\frac{T_{r}}{Y_{a}\delta} = (\Lambda)^{l_{l}\varepsilon-l} \xrightarrow{T_{l}\varepsilon-l} T_{l} = T_{a}\delta(\Lambda) = 4vvk$$

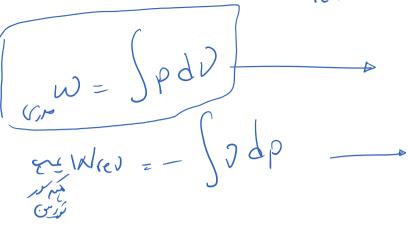


$$w_{i_0} = \int P dv$$

$$f_{i_0} = \int P dv$$

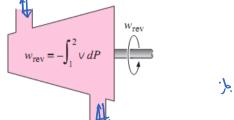
100

J

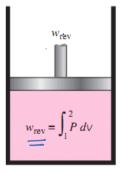


a kinetic and potential energies are ne

$$w_{\rm rev} = -\int_{1}^{2} v \, dP$$
 (kJ/kg)



(a) Steady-flow system



(b) Closed system

$$w_{rev} = -\left[\int v dp + bke + bpe\right]$$

$$(-bb) (D) =$$

$$W_{rev} = -\left[v\int_{0}^{x}dr + Dre + Dke\right]$$

 $W_{red} = -\left[2(P_r - P_i) + OPe + oke\right]$

ار به ل ترانی می ب م

Wrei

Advanced Termodynamic Page 4

$$= + \mathcal{V}(\mathcal{P}_{r} - \mathcal{P}_{r}) + \mathcal{J}(\mathcal{Z}_{r} - \mathcal{Z}_{r}) + \frac{1}{2} (\mathcal{V}_{r}^{r} - \mathcal{V}_{r}^{r})$$

$$= + \mathcal{V}(\mathcal{P}_{r} - \mathcal{P}_{r}) + \mathcal{J}(\mathcal{Z}_{r} - \mathcal{Z}_{r}) + \frac{1}{2} (\mathcal{V}_{r}^{r} - \mathcal{V}_{r}^{r})$$

$$= \mathcal{V}(\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r} = \mathcal{V}\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r} (\mathcal{K} - \mathcal{K}_{r}^{r})$$

$$= \mathcal{V}(\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r} = \mathcal{V}\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r} (\mathcal{K} - \mathcal{K}_{r}^{r})$$

$$= \mathcal{V}(\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r} = \mathcal{V}\mathcal{P}_{r} + \mathcal{G} + \frac{1}{2} \mathcal{V}_{r}^{r}$$

$$= \mathcal{V}(\mathcal{P}_{r} - \mathcal{P}_{r}) + \mathcal{O}(\mathcal{V}_{r} - \mathcal{V}_{r}^{r})$$

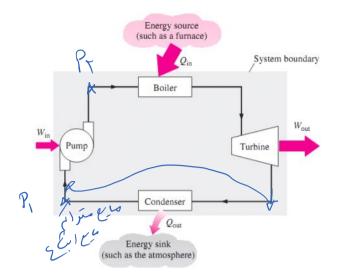
$$= \mathcal{V}(\mathcal{P}_{r} - \mathcal{V}_{r}) + \mathcal{O}(\mathcal{V}_{r} - \mathcal{V}_{r}^{r})$$

$$= \mathcal{V}(\mathcal{V}_{r} - \mathcal{V}_{r}) + \mathcal{O}(\mathcal{V}_{r} - \mathcal{V}_{r}^{r})$$

$$= \mathcal{V}(\mathcal{V}_{r} - \mathcal{V}_{r}) + \mathcal{O}(\mathcal{V}_{r} - \mathcal{V}_{r})$$

$$\begin{array}{l} \varphi_{Pero} \\ \varphi_{Kero} \\ \psi_{rev} \\ - \\ \int \mathcal{V} dP \\ = -\mathcal{V} \left(P_{I} - P_{I} \right) \\ \left(\varphi_{V} \right) \\ \mathcal{E} \\ \mathcal{U} \\$$

13



EXAMPLE 7-12 Compressing a Substance in the Liquid versus Gas Phases

Determine the compressor work input required to compress steam isentropically from 100 kPa to 1 MPa, assuming that the steam exists as (*a*) saturated liquid and (*b*) saturated vapor at the inlet state.

$$P_{i} = \log k \rho a \longrightarrow \rho_{i} \geq 1 \text{ M} \rho a$$

$$E_{i} \cup \bigcup \cup \bigcup i,$$

$$W_{red} = -\int \bigcup d\rho = -\Im \int_{i}^{r} d\rho = -\Im (\rho_{r} - \rho_{i})$$

$$W_{red} = -\int \bigcup d\rho = -\Im \int_{i}^{r} d\rho = -\Im (\rho_{r} - \rho_{i})$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a$$

$$P_{i} = \log k p a \qquad P_{i} = 1 \text{ M} p a \qquad P_$$

$g - \omega = h_t - h_1$ $w_{rev,in} = (3194.5 - 2675.0) \text{ kJ/kg} = 519.5 \text{ kJ/kg}$
$\mathcal{J} - \mathcal{W} = h_{t} - h_{t}$ $w_{rev,in} = (3194.5 - 2675.0) kJ/kg = 519.5 kJ/kg$
$h_{f} = h_{g} + \int_{257.5} h_{g} + \int_{2675.0} \int_{1.3028} \int_{0.0562} \int_{0.0562} \int_{0.3589} \int_{0.0562} \int_{0.056$
State 1: $P_1 = 100 \text{ kPa}$ $h_1 = 2675.0 \text{ kJ/kg}$ $s_1 = 7.3589 \text{ kJ/kg} \cdot \text{K}$ (Table A-5)
State 2: Thus, $ \begin{cases} P_2 = 1 \text{ MPa} \\ s_2 = s_1 \end{cases} $ $ h_2 = 3194.5 \text{ kJ/kg} \text{(Table A-6)} $